also applicable for cylindrical coordinates; in this case only the coefficient of  $p^*$  has to be changed in the system of equations.

The considerations in the logarithmic region do not directly concern the law of the wall presented here, since new findings in this area can be easily processed mathematically. The authors' suggestions concerning this problem are added to stimulate discussion of the universality of the constants in the logarithmic law of the wall.

### References

<sup>1</sup>Coantic, M., "Contribution à l'étude de la structure de la turbulence dans une conduite circulaire," Dissertation, Université d'Aix-Marseilles, Marseilles, France, 1966.

<sup>2</sup>Spalding, D. B., "A Single Formula for the Law of the Wall," *Journal of Applied Mechanics*, Vol. 28, Sept. 1961, pp. 455-457.

<sup>3</sup>van Driest, E. R., "On Turbulent Flow Near a Wall," Journal of the Aeronautical Sciences, Vol. 23, Nov. 1956, pp. 1007-1011.

<sup>4</sup>Szablewski, W., "Turbulente Grenzschichten in Ablösungsnähe," Zeitschrift für angewandte Matematik und Mechanik, ZAMM, Vol. 49, No. 4, 1969, pp. 215-225.

<sup>5</sup>Townsend, A. A., "Equilibrium Layers and Wall Turbulence," *Journal of Fluid Mechanics*, Nov. 1961, pp. 97-102.

<sup>6</sup>Reichardt, H., "Vollständige Darstellung der turbulente Geschwindigkeitsverteilung in glatten Leitungen," ZAMM, Vol. 31,

Geschwindigkeitsverteilung in glatten Leitungen," ZAMM, Vol. 31, July 1951, pp. 209-219.

<sup>7</sup>Eckelmann, H., "Experimentelle Untersuchungen in einer tur-

bulenten Kanalströmung mit starken viskosen Wandschichten," Göttengen, Max-Planck-Institut für Strömungsforschung, Nr. 48, 1970.

<sup>8</sup>Liu, C. K., Kline, S. J., and Johnston, S. P., "An Experimental Study of Turbulent Boundary Layers on Rough Walls," NSF Grant GP-2720, 1966.

<sup>9</sup>Nikuradse, J., "Gesetzmäßigkeiten der turbulenten Strömungin glatten Rohren," Forschungsheft 356, VDI Verlag, 1932.

<sup>10</sup>Coles, D. E. and Hirst, E. A., "Computation of Turbulent Boundary Layers," *Stanford Conference*, Vol. II, Stanford University, Calif., 1968, p. 5.

<sup>11</sup> Patel, V. C. and Head, M. R., "Some Observations on Skin Friction and Velocity Profiles in Fully Developed Pipe and Channel Flows," *Journal of Fluid Mechanics*, Vol. 38, Pt. 1, 1969, pp. 181-201

<sup>12</sup>Pfeil, H. and Göing, M., and Sticksel, W. J., "Betrachtung zum Wandgesetz," Jahrestagung der DGLR, Paper 81-017a, 1981.

<sup>13</sup> Pfeil, H. and Sticksel, W. J., "About the Influence of the Pressure Gradient on the Law of the Wall," AIAA Paper 81-0071, top 1081

 $^{14}$ Pfeil, H., Göing, M., "Studie über die Konstanten des Logarithmischen Wandgesetzes anhand von Meβergebnissen," Jahrestagung der DGLR, Paper 81-017b, 1981.

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# A Generalized Algebraic Stress Transport Hypothesis

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**S** OME 10 years ago, Rodi<sup>1</sup> and this writer<sup>2,3</sup> independently proposed ways of truncating a second-moment (or Reynolds stress) closure so that the differential transport equations for the Reynolds stresses  $\overline{u_i u_j}$  were replaced by a single transport equation for the turbulent kinetic energy k ( $\equiv \overline{u_i u_i}/2$ ) and a set of algebraic equations for the Reynolds stresses. Such closures have become widely used and are

commonly known as "algebraic stress models." The form adopted in Refs. 2 and 3 implies that the net transport of  $\overline{u_i u_j}$  denoted by  $T_{ij}$  (i.e., convection  $C_{ij}$  minus diffusion  $D_{ij}$ ) is related to the turbulent kinetic energy transport  $T_k$  by

$$C_{ij} - D_{ij} \equiv T_{ij} = \frac{2}{3} \delta_{ij} T_k \tag{1}$$

Mellor and Yamada<sup>4</sup> also obtained Eq. (1) following a different analytical path. Rodi<sup>1</sup> proposed that

$$T_{ij} = \frac{\overline{u_i u_j}}{k} T_k \tag{2}$$

Both forms contract properly and display the correct symmetry in i and j. However, while Eq. (1) gives no transport of the off-diagonal elements (i.e., the shear stresses), Eq. (2) suggests that shear stress transport proceeds just as readily as for the normal stress components. The latter postulate seems more reasonable and has been used by Rodi, <sup>1</sup> Gibson and Launder, <sup>5</sup> and others in the computation of turbulent free shear flows. In these studies the Reynolds stress transport equation is approximated as

$$T_{ij} = P_{ij} - \frac{2}{3}\delta_{ij}\epsilon - c_1\epsilon \left(\frac{\overline{u_i u_j}}{k} - \frac{2}{3}\delta_{ij}\right) - c_2\left(P_{ij} - \frac{1}{3}\delta_{ij}P_{kk}\right) \quad (3)$$

where  $\epsilon$  is the dissipation rate of turbulence energy and  $P_{ij}$  the generation rate of  $\overline{u_i u_j}$  by mean shear. A discussion of the physical basis of Eq. (3) is provided in Launder et al. <sup>6</sup>

Elimination of  $T_{ij}$  with Eq. (1) or (2) [noting that  $T_k$  equals  $(\frac{1}{2}P_{kk} - \epsilon)$ ] allows Eq. (3) to be rearranged to an algebraic equation for  $u_i u_i$ . For example, with Eq. (2) we obtain

$$\frac{\overline{u_i u_j} - \frac{1}{2} \delta_{ij} k}{k} = \frac{(I - c_2)}{(c_I - I + \lambda)} \left( P_{ij} - \frac{\delta}{3} ij P_{kk} \right) / \epsilon \tag{4}$$

where  $\lambda$  denotes  $P_{kk}/2\epsilon$  the local ratio of turbulent kinetic energy production to dissipation. For a two-dimensional thin shear flow (with x the flow direction and y the direction of principal gradient), Eq. (4) may be solved to give the following explicit formula for the shear stress

$$-\overline{uv} = \frac{2}{3} (1 - c_2) \frac{(c_1 - 1 + c_2 \lambda)}{(c_1 - 1 + \lambda)^2} \frac{k^2}{\epsilon} \frac{\partial U}{\partial y}$$

$$c_u(\lambda)$$
(5)

a form first given in Ref. 1.

The quantity  $c_{\mu}$  in Eq. (5) increases as  $\lambda$  tends toward zero. Figure 1 shows the variation reported in Gibson and Launder 5 where the constant coefficients  $c_1$  and  $c_2$  were given the values 2.2 and 0.55, respectively. The upward trend is, however, less rapid than that exhibited by the experimental data of far wakes and the wakes of self-propelled bodies collected by Rodi¹ whose recommended mean line also appears in Fig. 1. A further unsatisfactory feature of Eq. (5) is that on the axis or plane of symmetry of a jet,  $\lambda$  falls to zero, yet here experiments suggest that a value of  $c_{\mu}$  much closer to the local equilibrium ( $\lambda$ =1) value is called for.

In considering the cause of these disparities, note first that the off-diagonal Reynolds stresses are predominantly associated with the largest eddies present in the flow and, thus, the turbulent kinetic energy is associated with rather

Table 1 Values of  $c_{\mu}$  in round wake and plane jet

Flow	Experiment		Eq. (8)		Eq. (5)	
	Min	Axis	Min	Axis	Min	Axis
Round wake	0.45	0.56	0.45	0.53	0.2	0.25
Plane jet	0.08	0.12	0.10	0.19	0.10	0.25

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smaller scale motions than the shear stress. Now, the larger the scale of an eddy the longer will be its survival time and thus the greater its contribution to convective transport. We may thus conclude that the approximation for  $C_{ij}$  should allow a *preferential* transport of off-diagonal Reynolds stresses. This feature is readily achieved by combining elements of both Eqs. (1) and (2)

$$C_{ii} = C_k \left[ (1+\alpha) \left( \overline{u_i u_i} / k \right) - \alpha^2 / 3 \delta_{ii} \right]$$
 (6)

where  $\alpha$  is a positive coefficient. We adopt a precisely parallel representation of diffusive transport

$$D_{ij} = D_k \left[ (I + \beta) \left( \overline{u_i u_j} / k \right) - \beta^2 / 3 \delta_{ij} \right]$$
 (7)

Introduction of Eqs. (6) and (7) into Eq. (3) leads to the following implied formula for the coefficient  $c_n$ 

$$c_{\mu} = \frac{2}{3} \frac{(I - c_2) (c_1 + (I + \alpha) (\lambda - I) + (\alpha - \beta) \Lambda - (I - c_2) \lambda)}{(c_1 + (I + \alpha) (\lambda - I) + (\alpha - \beta) \Lambda)^2}$$

where  $\Lambda \equiv D_k/\epsilon$ , the ratio of the rates of diffusive gain of turbulence energy to viscous loss.

The choice  $\beta = \alpha$  produces the simplest stress-strain relation and, indeed, with these coefficients taken as 0.9 and retaining the above values for  $c_1$  and  $c_2$ , the variation of  $c_\mu$  with  $\lambda$  is very close to the *mean* experimental correlation (Fig. 1). This choice, however, exacerbates the problem of too high viscosities near a jet axis. Now, it must be said that no algebraic approximation is particularly satisfactory for representing diffusive transport—the use of Eq. (7) is justified mainly by the resulting overall success of the ASM formulation. If, therefore,  $\beta$  and  $\alpha$  are adjusted to allow Eq. (8) to match as well as possible the experimental behavior in the far round wake and in the jet,  $\beta$  needs to be negative because such a choice helps prevent  $c_\mu$  from rising excessively on the

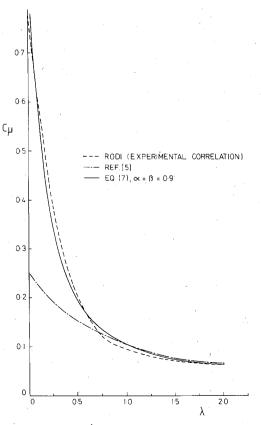


Fig. 1 Dependence of  $c_u$  on  $\lambda$ .

axis. Table 1 shows the values of  $c_{\mu}$  emerging for these two flows with  $\alpha=0.3$  and  $\beta=-0.8$  with values of  $\lambda$  and  $\Lambda$  taken from experiments cited by Rodi. Clearly Eq. (8) is more successful than Eq. (5) in capturing the observed variation of  $c_{\mu}$  in different flows. Further refinement in the choice of  $\alpha$  and  $\beta$  must await extensive numerical computations of these flows.

#### References

<sup>1</sup> Rodi, W., "The Prediction of Free Boundary Layers by Use of a Two-Equation Model of Turbulence," Ph.D. Thesis, University of London, London, Dec. 1972.

<sup>2</sup>Launder, B. E., "An Improved Algebraic Stress Model of Turbulence," Mechanical Engineering Dept., Imperial College, Rept. TM/TN/A8, 1971.

<sup>3</sup>Launder, B. E. and Ying, W. M., "Prediction of Flow and Heat Transfer in Ducts of Square Cross Section," *Proceedings of Institution of Mechanical Engineers*, Vol. 187, 1973, p. 455.

<sup>4</sup>Mellor, G. and Yamada, T., "A Hierarchy of Turbulence Closure Models for Planetary Boundary Layers," *Journal of Atmospheric Sciences*, Vol. 31, 1974, p. 1971.

<sup>5</sup>Gibson, M. M. and Launder, B. E., "On the Calculation of Horizontal Free Shear Flows under Gravitational Influence," *Journal of Heat Transfer*, Vol. 98C, 1976, p. 80.

<sup>6</sup>Launder, B. E., Reece, G. J., and Rodi, W., "Progress in the Development of a Reynolds-Stress Turbulence Closure," *Journal of Fluid Mechanics*, Vol. 68, 1975, p. 537.

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# Effect of Thickness on Airfoil Surface Noise

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## Introduction and Discussion of Existing Theory

THE noise radiated from the fixed and rotating surfaces of an aircraft is a matter of concern. The experimental results of this paper can be applied in these areas through appropriate theories describing the noise from an airfoil (e.g., fan, helicopter rotor, flap, and wing noise).

The early airfoil noise theories (e.g., Ref. 1) were for incompressible flow about a thin, small-chord airfoil. The thin airfoil does not disturb the straight mean flow streamlines. Goldstein and Atassi<sup>2</sup> accounted for the real airfoil effects of thickness, camber, and angle of attack for a small-chord airfoil in incompressible flow. These real airfoil effects modify the straight mean flow streamlines which thereby distorts the incident turbulent gusts. They suggest that this distortion will affect the radiated noise, especially at high frequency. Goldstein also formulated fundamental theories that account for the major effects of transverse mean velocity gradients (e.g., jet noise in Ref. 3, and the noise from the leading and trailing edges of a thin infinite plate in Ref. 4). These theories were shown by Olsen to be quite accurate. 5-7 Goldstein also showed (Ref. 1, pp. 137-145) that the early airfoil theory (incompressible flow over a small chord-thin airfoil) is easily modified to include the effect of compressibility. According to Ref. 1, this compressible theory can also be used to predict the spectra at 90 deg (i.e., normal to plane of airfoil) for airfoils of finite chord immersed in a mean velocity gradient. Olsen<sup>8</sup> demonstrated experimentally that this theory predicts the spectra at 90 deg for finite-chord

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